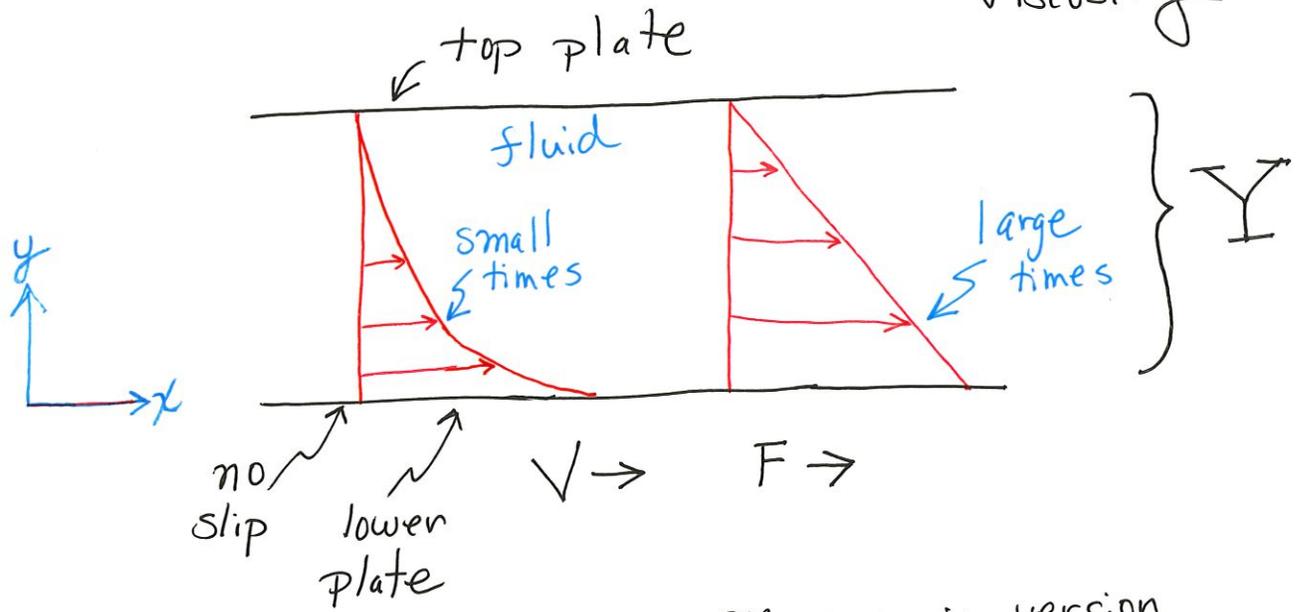


ENCH 630 - Lecture 2: Newton's Law of Viscosity



$$\frac{F}{A} = \mu \frac{V}{Y}$$

Shear force per unit area = shear "stress"
 ← viscosity
 ← Macroscopic version of Newton's law of viscosity
 ← velocity gradient

Microscopic version:

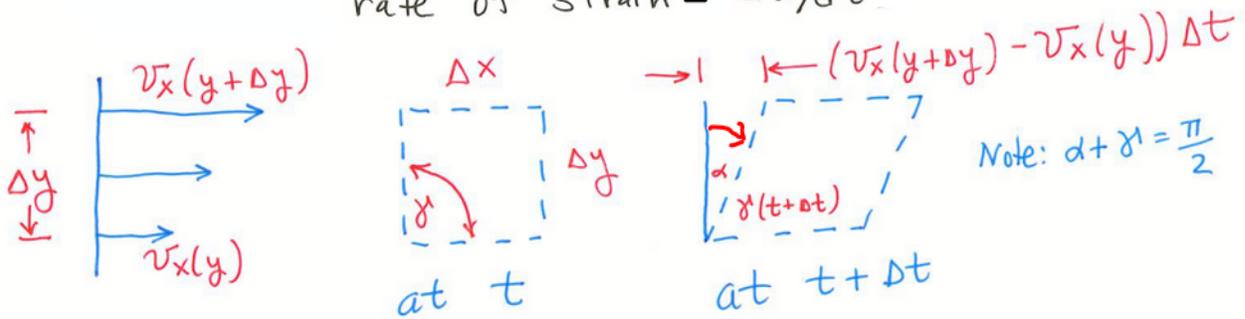
$$\tau_{yx} = -\mu \frac{dV_x}{dy}$$

Shear stress in x direction on a plane of constant y
 ← Momentum transferred in direction opposite to gradient (from high to low velocity)
 ← rate of strain

Units ?

For fluids, the material continuously deforms when stress is applied.

$$\text{rate of strain} = -d\alpha/dt$$



$$\begin{aligned} \frac{d\alpha}{dt} &= \lim_{\Delta t \rightarrow 0} \frac{\alpha(t+\Delta t) - \alpha(t)}{\Delta t} \\ &= \lim_{\Delta x, \Delta y, \Delta t \rightarrow 0} \left[\frac{\tan^{-1} \left\{ \frac{(v_x(y+\Delta y) - v_x(y)) \Delta t}{\Delta y} \right\} - 0}{\Delta t} \right] \end{aligned}$$

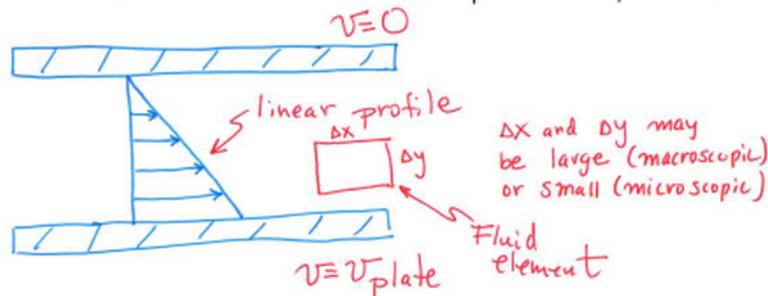
For small α , we have $\tan^{-1}(x) = x$ so that

$$\frac{d\alpha}{dt} = -\frac{d\alpha'}{dt} = \frac{dv_x}{dy} = -\text{rate of strain}$$

Which also means

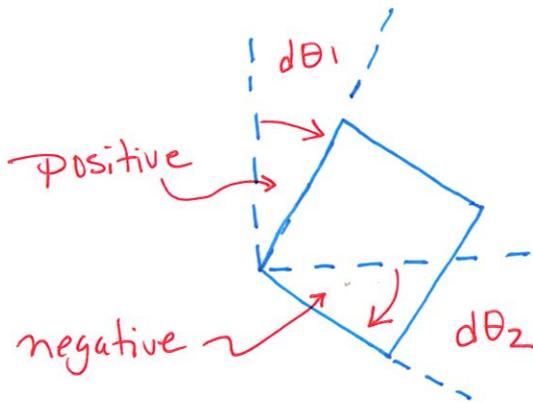
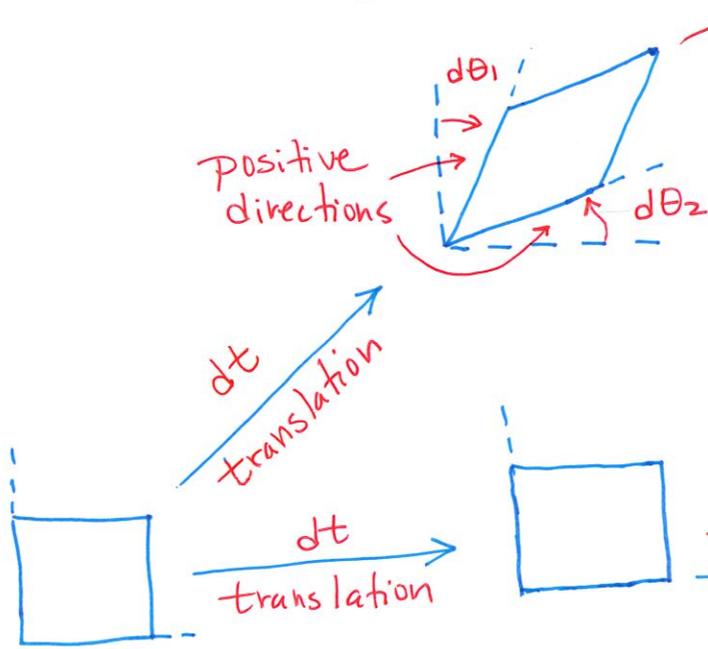
$$\mu = \text{shear stress} / \text{rate of strain}$$

Does $\tau_{yx} = \tau_{xy}$ apply? Consider steady state and fully developed flow between parallel plates.



We know in this a fluid element regardless of size does not accelerated either linearly or angularly. Therefore τ_{yx} and τ_{xy} are independent of position and $\tau_{yx} = \tau_{xy}$. A more complex argument can be used to show $\tau_{yx} = \tau_{xy}$ even for unsteady flow.

More generally, we have the following



$$\frac{d\theta_1}{dt} + \frac{d\theta_2}{dt} > 0$$

(rate of strain $\neq 0$)

$$\frac{d\theta_1}{dt} - \frac{d\theta_2}{dt} = 0$$

(no rotation)

$$\frac{d\theta_1}{dt} + \frac{d\theta_2}{dt} = 0$$

(rate of strain = 0)

$$\frac{d\theta_1}{dt} - \frac{d\theta_2}{dt} > 0$$

(no rotation)

$$\frac{d\theta_1}{dt} + \frac{d\theta_2}{dt} = 0$$

(rate of strain = 0)

$$\frac{d\theta_1}{dt} - \frac{d\theta_2}{dt} > 0$$

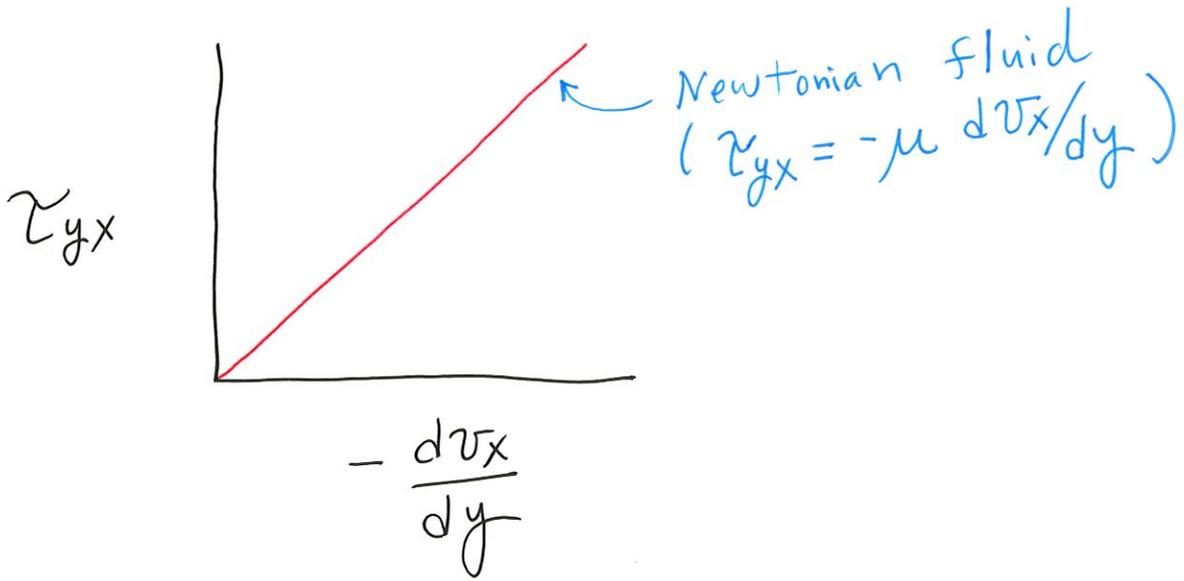
(rotation)

Using arguments similar to before, and assuming $\frac{\partial v_x}{\partial y} \neq 0$ and $\frac{\partial v_y}{\partial x} \neq 0$ we get:

$$\text{Total rate of strain} = -\frac{d\theta_1}{dt} - \frac{d\theta_2}{dt} = -\frac{\partial v_x}{\partial y} - \frac{\partial v_y}{\partial x}$$

$$\text{Rate of rotation (equal to average of the rotation rates of the two sides): } \omega = \frac{1}{2} \left(\frac{d\theta_1}{dt} - \frac{d\theta_2}{dt} \right) = \frac{1}{2} \left(\frac{\partial v_x}{\partial y} - \frac{\partial v_y}{\partial x} \right)$$

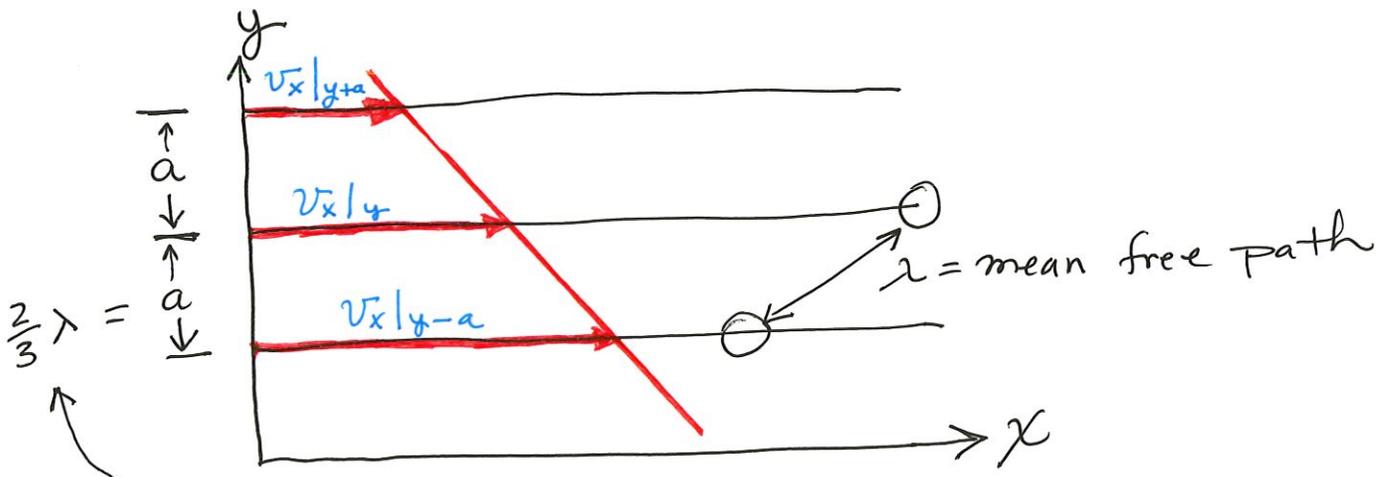
Types of fluids



Theories of viscosity: Kinetic theory of gases.

1. Maxwell Boltzmann distribution of velocities
2. Molecules do not interact until they collide, then they collide elastically

Potential energy diagram?



average length parallel to y axis travelled between collisions

Kinetic theory results:

Average molecular speed:

$$\bar{u} = \sqrt{\frac{8kT}{\pi m}}$$

Boltzmann constant = R/N_A

molecule mass

Frequency of bombardment
Per unit area:

$$z = \frac{1}{2} \bar{u} n$$

number density of molecules

Mean free path:

$$\lambda = \frac{1}{\sqrt{2} \pi d^2 n}$$

diameter of molecule

Molecules arriving at a plane have on average had their last collision at a distance $\frac{2}{3}$ of the mean free path from the wall (denote as "a")

$$a = \frac{2}{3} \lambda$$

By definition (using momentum flux definition), and assuming that molecules arriving at plane y have x -direction momentum representative of the region they last collided:

$$\tau_{yx} = z(m v_x|_{y-a}) - z(m v_x|_{y+a})$$

but

$$v_x|_{y-a} = v_x|_y - \frac{2}{3} \lambda \frac{dv_x}{dy}$$

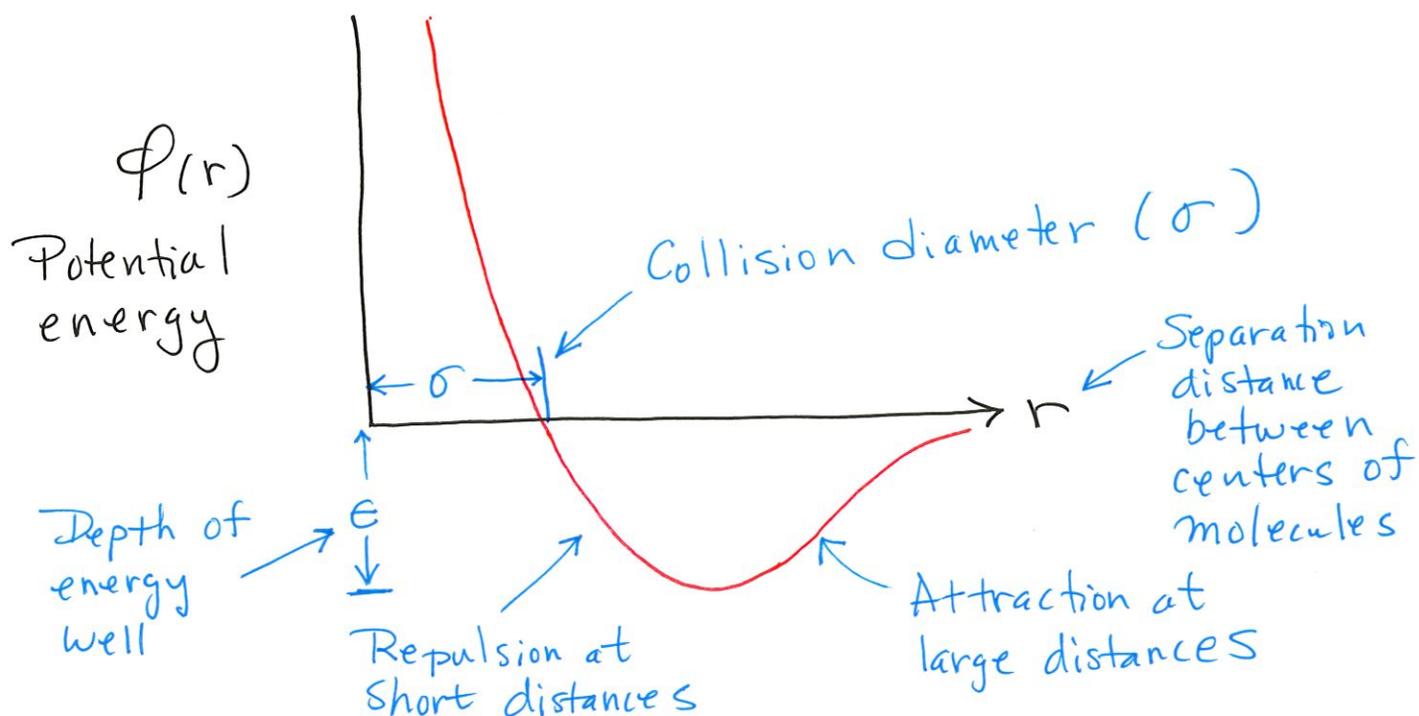
$$v_x|_{y+a} = v_x|_y + \frac{2}{3} \lambda \frac{dv_x}{dy}$$

So that

$$\tau_{yx} = -\frac{1}{3} n m \bar{u} \lambda \frac{dv_x}{dy}$$

$$\frac{2}{3} \frac{1}{\pi^{3/2}} \frac{\sqrt{mkT}}{d^2} \leftarrow \mu$$

More rigorous theory



Leonard Jones (6-12) potential

$$\phi(r) = 4\epsilon \left[\left(\frac{\sigma}{r}\right)^{12} - \left(\frac{\sigma}{r}\right)^6 \right]$$

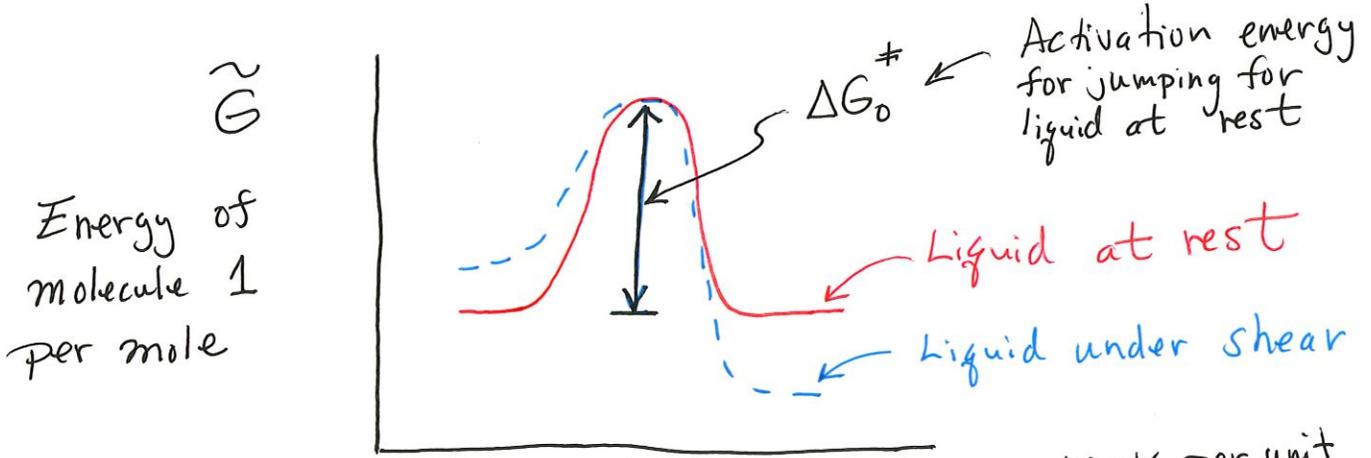
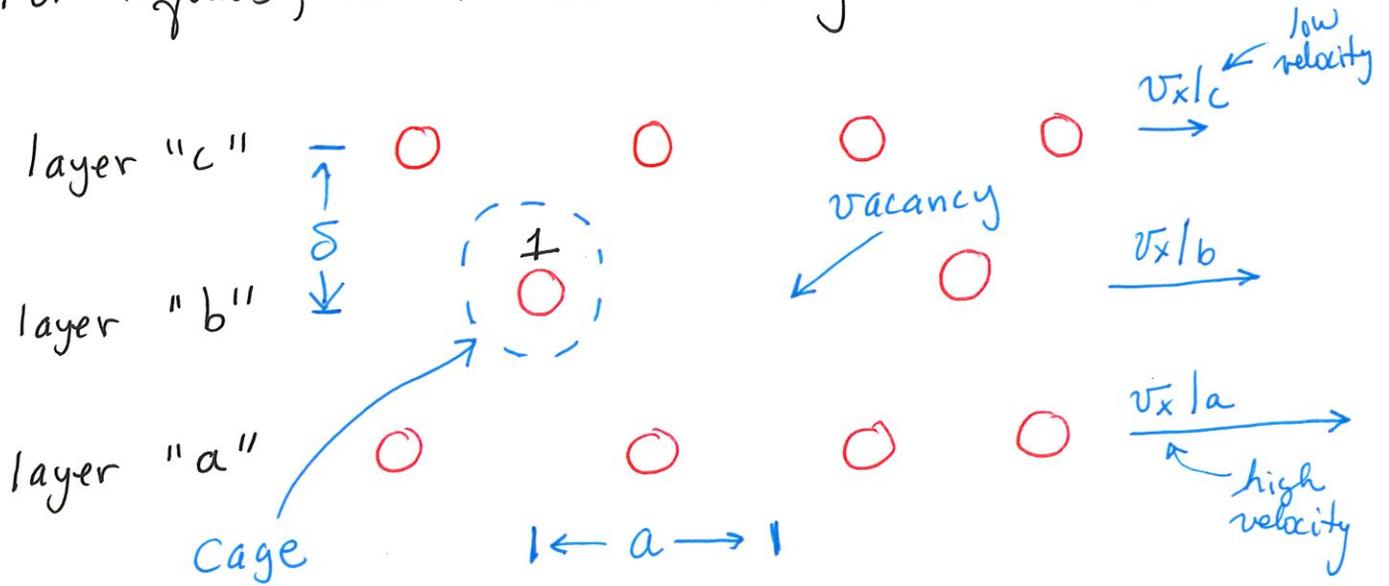
Leads to Chapman-Enskog theory

$$\mu = 2.669 \times 10^{-5} \frac{\sqrt{MT}}{\sigma^2 \Omega}$$

$\xrightarrow{\text{g/cms}}$ $\overset{\circ}{\text{A}} \rightarrow$ $\xrightarrow{\text{molecular weight}}$

Where Ω = function of kT/ϵ

For liquids, a "lattice" theory can be used



Easier to jump right and harder to jump left

$$\Delta \tilde{G}^\ddagger = \Delta \tilde{G}_0^\ddagger \pm \underbrace{\frac{a}{2}}_{\text{Distance from start to top of barrier}} \underbrace{\frac{\tau_{yx}}{\delta}}_{\text{shear force per area / } \delta = \text{shear force per unit volume}} \underbrace{\tilde{V}}_{\text{Volume per } m_0}$$

Work per unit volume

Frequency of forward and backward jumps

$$k_f = \frac{kT}{h} \exp(-\Delta G_0^\ddagger / RT) \exp\left(\frac{a \tau_{yx} \tilde{V}}{2 \delta RT}\right)$$

$$k_b = \frac{kT}{h} \exp(-\Delta G_0^\ddagger / RT) \exp\left(-\frac{a \tau_{yx} \tilde{V}}{2 \delta RT}\right)$$

The net velocity by which layer b slips ahead of layer c is given by

$$v_x|_b - v_x|_c = a (k_f - k_b)$$

Divide by δ :

$$\frac{dv_x}{dy} = \frac{a}{\delta} (k_f - k_b)$$

Substitute, assuming $\frac{a \tilde{\tau}_{yx} \tilde{V}}{2 \delta RT}$ is small

$$\tilde{\tau}_{yx} = - \underbrace{\left(\frac{\delta}{a} \right)^2 \frac{N_{av} h}{\tilde{V}} \exp\left(\frac{\Delta G_o^\ddagger}{RT} \right)}_{\mu} \frac{dv_x}{dy}$$

Approximately: $\Delta G_o^\ddagger \approx 0.4 \Delta \tilde{U}_{\text{vaporization}}$

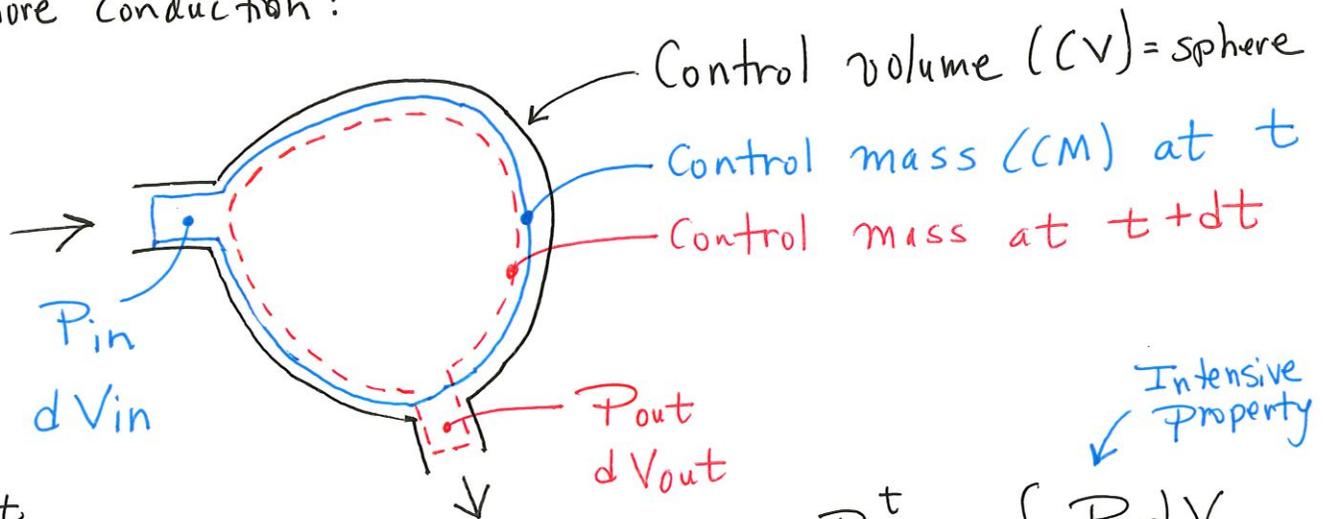
Also, Trouton's Rule: $\Delta \tilde{U}_{\text{vaporization}} \approx 9.4 R T_b$

Normal boiling point

The Reynolds Transport Theorem

Convert conservation law for control mass (closed) to control volume (open) system

Ignore conduction:



P^t = any extensive property;

$$P_{CV}^t = \int_{CV} P dV$$

$$P_{CM}^t = \int_{CM} P dV$$

$$P_{CM}^t |_{t} = P_{CV}^t |_{t} + P_{in} dV_{in}$$

$$P_{CM}^t |_{t+dt} = P_{CV}^t |_{t+dt} + P_{out} dV_{out}$$

Therefore:

$$\frac{dP_{CM}^t}{dt} = \frac{dP_{CV}^t}{dt} - P_{in} \left(\frac{dV}{dt} \right)_{in} + P_{out} \left(\frac{dV}{dt} \right)_{out}$$

↑ Accumulation in CV
 ↑ Inflow to CV
 ↑ Outflow from CV

In general, if $\frac{dP_{CM}^t}{dt} = B$ for a control mass, then for a control volume production of $P_{CV}^t = B$

Examples ?

Including conduction (when P = momentum/volume):
 Include conduction in the "B"
 Include conduction in the "ins-outs" - 9 -